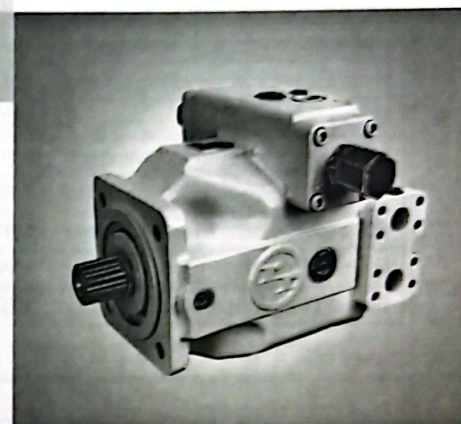


Variable displacement pump A4VSG

RE 92 100/11.95 1/36
Replaces: 01.95

closed circuit

Sizes 40...1000
Series 1 and 2
Nominal pressure 350 bar
Peak pressure 400 bar



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Features

The axial piston swashplate design variable displacement pump A4VSG is designed for hydrostatic transmission in closed circuit.

Flow is proportional to input speed and displacement and is infinitely variable by adjustment of the swashplate.

- slot-controlled swashplate design
- infinitely variable adjustment of displacement
- reversible flow
- permissible nominal pressure 350 bar
- low noise level
- long service life
- drive shaft capable of absorbing axial and radial loads
- high power/weight ratio
- modular design
- short control times
- through drive and tandem pumps possible
- pump swivel angle indicator
- installation position optional
- operation on HF fluids possible with reduced operating parameters

For description of control and regulating devices see separate RE sheets

RE 92055, RE 92060, RE 92064
RE 92072, RE 92076, RE 92080

Fluid

For extensive information on the selection of fluids and for application conditions, please consult our data sheets RE 90220 (mineral oils), RE 90221 (environmentally acceptable fluids) or RE 90223 (HF fluids) before proceeding with the design stage. When operating with environmentally acceptable or HF fluids reduced operating conditions may apply.

Operating viscosity range

In order to obtain optimum efficiency and service life, we recommend that the operating viscosity (at operating temperature) be selected from within the range:

$$v_{opt} = \text{operating viscosity } 16...36 \text{ mm}^2/\text{s}$$

referred to the closed loop temperature.

Viscosity limits

The limiting values for viscosity are as follows:

$$v_{min} = 10 \text{ mm}^2/\text{s}$$

short term at a maximum permissible drain temperature of 90° C.

$$v_{max} = 1000 \text{ mm}^2/\text{s}$$

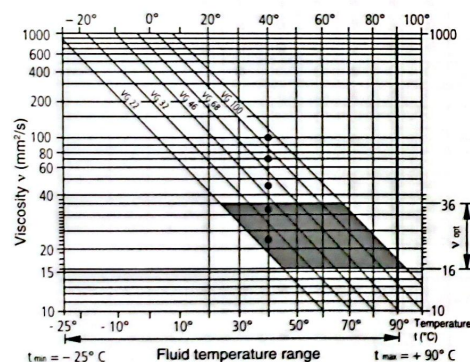
short term on cold start.

Notes on the selection of hydraulic fluid

In order to select the correct fluid, it is necessary to know the operating temperature in the circuit (closed loop), in relation to the ambient temperature.

The hydraulic fluid should be selected so that within the operating temperature range the operating viscosity lies within the optimum range (n_{opt}), (see shaded section of selection diagram). We recommend that the highest possible viscosity range be chosen in each case.

Selection diagram



Example: At an ambient temperature of X° C the operating temperature is 60° C. Within the optimum operating viscosity range (n_{opt} ; shaded area) this corresponds to viscosity ranges VG 46 or VG 68. VG 68 should be selected.

Important: The leakage oil (case drain oil) temperature is influenced by pressure and speed and is always higher than the circuit temperature. However at no point in the circuit may the temperature exceed 90° C.

If it is not possible to comply with the above conditions because of extreme operating parameters or high ambient temperatures, please consult us.

Flushing of the bearings

With the following operating conditions the bearings should be flushed to ensure correct functioning over a long period:

- with special fluids (not mineral) due to limited lubricity and a narrow operating temperature range
- when operating with mineral oils in limited conditions of temperature and viscosity
- with vertical installation (drive shaft facing upwards) flushing of the bearings is recommended for lubrication of the front bearing and shaft sealing ring.

Flushing of the bearings is carried out via port "U" in the vicinity of the front flange of the variable pump. The flushing oil flows through the front bearing and out with the pump case drain oil at the drain port.

The following quantities are required for flushing the various sizes:

Size	40	71	125	180	250	355	500	750	1000
Q_{sp} L/min	3	4	5	7	10	15	20	30	40

For the given flushing quantities there will be a pressure difference of approx. 2 bar between port "U" (including fittings) and the case drain oil chamber.

Filtration of fluid (axial piston unit)

In order to ensure correct functioning of the unit, a minimum level of cleanliness to NAS 16389 class 9

SAE class 6

ISO/DIS 4406 class 18/15 is necessary.

This is achievable for example with a filter element

Type...D 020...(see RE 31278).

This gives a filter quotient of

$$\beta_{20} \approx 100$$

If a filter for the boost circuit is factory mounted (Ordering code [F]), depending on the size of the axial piston unit the following filters are installed, fitted with opto-electrical clogging indicator as standard:

Sizes 40 and 71: LFBN/HC60G20D1.0/24/V
 Sizes 125, 180 and 250: LFBN/HC110G20D1.0/24/V
 Size 355: LFBN/HC240G20D1.0/L24/V
 Size 500: LFBN/HC330G20D1.0/L24/V

For further details see RE 31278.

Temperature range (cf. selection diagram)

$t_{min} = -25^\circ \text{C}$

$t_{max} = +90^\circ \text{C}$

Installation instructions

Installation position:

Optional. The pump housing must be filled with hydraulic fluid when commissioning and during operation.

In order to minimise noise levels, all connecting piping (suction, pressure, case drain oil ports) must be disconnected from the tank by means of flexible elements.

The use of check valves in the case drain oil line is to be avoided. They, however, may be used in certain cases after consultation with us.

Technical data

(applicable for operation with mineral oils)

Operating pressure range - Inlet side

Recommended boost pressure p_{sp} 16 bar

Recommended boost pressure with common auxiliary pump for boost and pilot oil circuits (EO1) p_{sp} 25 bar

Maximum boost pressure – auxiliary pump max. pressure P_{Hmax}

for MA-, EM-, HM-, HS-, EO-, DS-settings 50 bar
 for HD-, HW-settings and LR.N- and DR-control 16 bar

Auxiliary pumps - inlet pressure

Suction pressure p_{smin} ($v = 10...300 \text{ mm}^2/\text{s}$) $\geq 0,7$ bar absolute

Operating pressure range - outlet side

(pressures to DIN 24312)

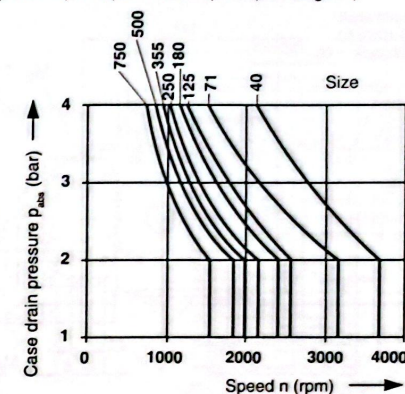
Pressure at port A or B

Nominal pressure p_N 350 bar

Peak pressure p_{max} 400 bar

Case drain pressure

The maximum permissible case drain pressure (housing pressure) is dependent on speed (see diagram).



Max. case drain pressure (housing pressure)

p_{Lmax} 4 bar

These are approximate values. Under certain operating conditions a reduction in these values may be necessary.

Table of values (theoretical values, without considering η_{mh} and η_v ; values rounded)

Size			40	71	125	180	250	355	500	750	1000	
Displacement	$V_{g\ max}$	cm ³	40	71	125	180	250	355	500	750	1000	
Max. speed	n_{max}	rpm	3700	3200	2600	2400	2200	2000	1800	1600	1600	
Max. flow	at n_{max}	Q_{max}	L/min	148	227	325	432	550	710	900	1200	1600
	at $n_E = 1500$ rpm		L/min	60	107	186	270	375	533	750	1125	1500
Max. power	at $n_{o\ max}$	$P_{o\ max}$	kW	86	132	190	252	321	414	525	700	933
($\Delta p = 350$ bar)	at $n_E = 1500$ rpm		kW	35	62	109	158	219	311	438	656	875
Max. torque ($\Delta p = 350$ bar)	at $V_{g\ max}$	T_{max}	Nm	223	395	696	1002	1391	1976	2783	4174	5565
Torque ($\Delta p = 100$ bar)	at $V_{g\ max}$	T	Nm	64	113	199	286	398	564	795	1193	1590
Moment of inertia about drive axis		J	kgm ²	0,0049	0,0121	0,03	0,055	0,0959	0,19	0,3325	0,66	1,20
Filling volume		L	L	2	2,5	5	4	10	8	14	19	27
Approx. weight (pump with EO1 control and valve block)	m	kg	47	60	100	114	214	237	350	500	630	
Max. axial force	$\pm F_{ax\ max}$	N	600	800	1000	1400	1800	2000	2000	2200	2200	
Max. radial force	$F_{q\ max}$	N	1000	1200	1600	2000	2000	2200	2500	3000	3500	

Determination of size

$$\text{Output flow } Q = \frac{V_g \cdot n \cdot \eta_v}{1000} \quad [\text{L/min}]$$

$$\text{Torque } T = \frac{1,59 \cdot V_g \cdot \Delta p}{100 \cdot \eta_{mh}} \quad [\text{Nm}]$$

$$\text{Power } P = \frac{2\pi \cdot T \cdot n}{60000} = \frac{T \cdot n}{9549} = \frac{Q \cdot \Delta p}{600 \cdot \eta_t} \quad [\text{kW}]$$

V_g = geometr. displacement [cm³] per revolution

Δp = Differential pressure [bar]

n = Speed [rpm]

η_v = Volumetric efficiency

η_{mh} = Mechanical-hydraulic efficiency

η_t = Total efficiency ($\eta_t = \eta_v \cdot \eta_{mh}$)

Application of forces

